

Slide Screws/Trapezoidal Screws/Components for Trapezoidal Screws

Slide Screws Trapezoidal Screws Components for Trapezoidal Screws

Product Name	Miniature Slide Screws—Straight—	Nut Brackets	—One End Stepped / Both Ends Stepped—	—One End Double Stepped / One End Stepped, One End Double Stepped—
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Information

30 Degree Trapezoidal Screw Threads - One End Triple Stepped Type and Both Ends Triple Stepped Type - and Stop Plate Sets are listed on our website.

<http://fa.misumi.jp/hp-item.jsp>

These products are still available. Visit the web site above.(Only in Japanese)



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Trapezoidal Screws Technical Data

■Trapezoidal Screw Technical Calculation

1. Selection of Nut Material

(1) Contact Pressure P (N/mm²)

$$P = \frac{F_s}{F_o} \times \alpha$$

F_s: Axial Load (N)
F_o: Allowable Dynamic Thrust (N)
α: α=9.8 (Brass), 0.98 (Resin)

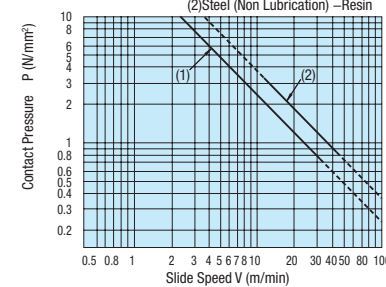
(Allowable Dynamic Thrust: The thrust when the contact pressure acting on the thread shaft and nuts is 9.8N (0.98N)/mm²)

(2) Slide Speed V (m/min)

$$V = \frac{\pi \cdot d_2 \cdot n}{\cos(\delta)} \times 10^{-3}$$

d₂: Effective Dia. of Thread Shaft (mm)
n: Thread Shaft Revolution Frequency per Minute (min⁻¹)
δ: Thread Shaft Lead Angle (Degree)

PV Value Graph



2. Screw Efficiency and Generated Thrust

(1) Screw Efficiency η

$$\eta = \frac{1 - \mu \tan(\delta)}{1 + \mu / \tan(\delta)}$$

μ: Dynamic Friction Coefficient
δ: Thread Shaft Lead Angle (Degree)

(2) Generation Thrust Fa (N)

$$F_a = \frac{2\pi \cdot \eta \cdot T}{R} \times 10^3$$

η: Screw Efficiency
T: Input Torque (N·m)
R: Lead (mm)

■Features of Oil Free Nut for 30 Degree Trapezoidal Screw Thread

MISUMI oil free nut has the same effect as that of greased nut refueled every 45 hours, as shown in the abrasion result on the right. After initial greasing, this realizes oil-free operation. It presents high performance especially under low-speed operation.

Calculation Example

When using thread shaft MTSR16 and nut MTSFR16 for axial load 300 (N) and thread shaft rotary speed 500min⁻¹

(1) Contact Pressure P (N/mm²)

$$P = \frac{F_s}{F_o} \times \alpha = \frac{300}{6670} \times 9.8 = 0.44 \text{ (N/mm}^2\text{)}$$

(2) Slide Speed V (m/min)

$$V = \frac{\pi \cdot d_2 \cdot n}{\cos(\delta)} \times 10^{-3} = \frac{\pi \times 14.5 \times 500}{\cos(3^\circ 46')} \times 10^{-3} = 22.8 \text{ (m/min)}$$

See the calculated value of P and V from the PV value graph, the value is less than allowable speed when P=0.44 (N/mm²). This indicates the fact that abnormal abrasion cannot occur.

Calculation Example

Required torque when thread shaft MTSR16 and nut MTSFR16 are used.

(1) Screw Efficiency η

$$\eta = \frac{1 - \mu \tan(\delta)}{1 + \mu / \tan(\delta)} = \frac{1 - 0.21 \times \tan(3^\circ 46')}{1 + 0.21 / \tan(3^\circ 46')} = 0.24$$

(2) Generation Thrust Fa (N)

$$F_a = \frac{2\pi \cdot \eta \cdot T}{R} \times 10^3 = \frac{2\pi \times 0.24 \times 8}{3} \times 10^3 = 4.02 \text{ (kN)}$$

When calculating required torque T (N·m) to obtain thrust Fa 4.02 (kN).

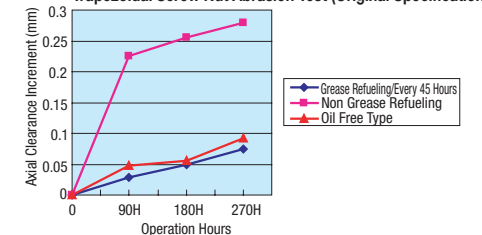
(3) Required Torque T (N·m)

$$T = \frac{F_a \cdot R}{2\pi \cdot \eta} \times 10^{-3} = \frac{4.02 \times 10^3 \times 3}{2\pi \times 0.24} \times 10^{-3} = 8 \text{ (N·m)}$$

■Reference Value

Thread	Nuts	Dynamic Friction Coefficient μ
Steel (Lubrication)	Brass	0.21
Steel (Non Lubrication)	Polyacetal/PPS Resin with Sliding Property	0.13

Trapezoidal Screw Nut Abrasion Test (Original Specification)



Test Conditions: Sample Thread Shafts: MTSRW10-270 Nut: MTSFR10 · MTSMR10 (Oil Free Type)
Movable Load 220N Shaft Revolution Frequency 1500rpm